Simulation and Optimization Studies on the Ring Rolling Process
Using Steel and Aluminum Alloys

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Abstract. The current research was carried out using ANSYS to optimize the process parameters for the ring rolling process. In order to optimize the ring rolling process, parameters such as speed, axial roller feed, and driving rollers have been assessed. As a process optimization approach, the optimum values of the parameters and their relationships need to be evaluated. The stress and strain levels were evaluated at various speeds and forces and the critical failure values were determined. The structural steel and aluminum alloys were chosen for this research because they are used as a roller and job part components in the solid wheels for locomotive applications, respectively. The study was conducted by varying the guide roller's angular velocity from 40 to 45 rad/sec and varying the work piece’s angular velocity from 200 to 250 rad/sec. Additionally, the work part and roller’s fatigue strengths were determined based on the number of cycles before failure. To evaluate the stresses of plastic strain and von failures, the full stress analysis was also performed.

Keywords: roller, ANSYS, workpiece, plastic strain, von Misses stress, metal alloy.

1 Introduction

Rolling is a continuous metal shaping between a series of spinning or rotating rolls, whose shape or height is gradually decreased to produce the desired segment by applying strong plastic deformation pressures. It is the method of thickness reduction which reduces the length without significantly increasing the width. The ring rolling method can be carried out at high temperature (hot) or at ambient temperature (cold) with the product initially. Ring rolling is an innovative technique in the development of smooth rings of elastic cross-sectional form, enhanced grain structure, and reduced scrap. The ring is formed by a local continuum rolling method as shown in the figure from an initial void, incrementally from a small diameter and thick section to a wide diameter and thin section. In the area of metal plastic manufacturing, research and development of ring rolling techniques with rings complicated in form or wide in length or with high precision have become an important topic. Because of the process’s nature and strong non-linearity, it is difficult to describe the process correctly by analytical methods alone. Although quantitative explanations are appropriate for the system they are being based on, findings are challenging to extrapolate accurately. The finite element approach for researching and developing innovative ring rolling techniques is, therefore, inspired [1]. Developing a realistic 3D finite-element ring rolling model has become an urgent issue, and the problem of how to properly monitor guide rolls is one of the key issues in achieving a good 3D finite-element ring rolling simulation, particularly for rings that are complicated in shape or wide in size or with high precision [2].

2 Literature Review

Some journal papers were selectively studied which have direct relevance with the current research work and their results are discussed here.

Yu-Min Zhao explained ring rolling as an advanced technique of plastic forming used in the production of precise seamless rings. The rolling ratio is a decisive parameter for the rolling process as it determines the blank dimension, reflects the degree of ring deformation and influences the results of the forming. Nonetheless, the importance of rolling is always overlooked. In this paper, the impact of the rolling ratio on the rolling of the groove-section profile circle is explored. The relationships between rolling ratio and blank length, rolling ratio and degree of ring deformation are theoretically analyzed and a rolling ratio conceptual quality array is proposed [3]. Then the influences of the rolling ratio on the forming results are revealed with the simulation of finite ele-
ments. A fair quality scope of the rolling ratio is calculated based on comparison and observational validation.

To simulate the non-linear problem that characterizes the ring rolling process, finite element codes are available [4]. The analysis is done using ANSYS. The parameters including speed, axial roller feed, and driving rollers were analyzed from the study [5]. The optimal relationship between the process parameters is identified from the analysis. The stress and strain values are generated at different speeds. Critical and failure values are also obtained. The research was carried out for structural steel and aluminum as the workpiece, taking the characteristics of the metal into the account [6].

3 Research Methodology

3.1 Process of ring rolling

The design scheme of the ring rolling process is presented in Figure 1.

![Figure 1 – The ring rolling process](image1)

Two rollers that rotate in the opposite direction are fed in the ring rolling material. The gap between the rollers is lower than the material thickness caused by deformation. It is caused to elongate due to reduced material thickness. Material-roller resistance induced the material to move [2]. The volume of deformation in a single pass is limited by roller friction. If the thickness varies, rollers can be slipped. Certain procedures such as shearing, flattening and punching are to be completed before the ring rolling process in order to generate the final product. Shearing is the method of removing a necessary blank from the stock. Flattening is a method of adding sufficient force to reduce the height of the original element [5]. Eventually, the punch and dies design creates a gap in the part [7]. The processing of ring rolling is given in Figure 2

3.2 Typical Ring Rolling Products

Rolled rings find application in bearings, slewing bearings, turbine disks and gear blanks [8]. Ring rolling machines are also used in producing solid wheels and wheel disks for high-speed trains, locomotives, railway carriages, trams, and subway trains. More examples of the varied uses of ring rolling items include bevel gear and axle drive wheels for the automotive industry, transmission manufacturing, turbine manufacturing (turbine disks for plane propulsion engines), flanges in the computer and plant construction industry, rings for tower flanges (in off-shore wind turbines) and roller bearings (cold spinning) [9].

![Figure 2 – The process of ring rolling](image2)

3.3 Analytical description of ring rolling

Kalpak Jian and Schmidt’s flat rolling analysis is extended to the process of ring rolling [9]. Process parameters are defined as given in Figure 3.

![Figure 3 – Description of the ring rolling process](image3)

The parameters used in the analysis are $d_i$ – inner diameter; $d_o$ – outer diameter; $d_r$ – roll diameter; $d_m$ – mandrel diameter; $n_r$ – roller rotational speed; $n_m$ – ring rotational speed; $v_o$ – advance velocity of the mandrel.

The first relationship to be developed is the dependency by volume preservation between cross-sectional thickness and diameter. The plain strain is presumed in this situation, therefore there is no strain in the width direction.

The main geometric dependencies are based on the following requirement of the constant volume:

$$\pi(d_r^2 - d_i^2)w/4 = \pi(d_o^2 - d_i^2)w/4;$$

$$d_i = [d_r^2 - (d_o^2 - d_i^2)]^{1/2}. \quad (1)$$

Therefore, the internal diameter relies on the external diameter and the actual empty volume as determined from the initial measurements of the circle. The next step in the analysis is to provide equivalence to the process of flat rolling by equating the contact lengths between the material and the roll or mandrel. This analysis targets defining the equivalent diameter of a flat rolling process roll to represent the more complex curvilinear ring roll.
A result for the forming roll which undergoes convex-convex contact is given by

$$d_{eq} = d_r[1 + 2d_r/(d_{m0} + d_m)].$$

Therefore, the corresponding flat rolling size is less than the true diameter of the greater convex-convex touch roll. Similarly, the diameter of a mandrel is given by

$$d_{m,eq} = d_r[1 - 2d_r/(d_{m0} + d_m)].$$

The conversion of the convex-concave touch of the mandrel to the inner ring surface leads the corresponding flat roll size to increase than convex-flat contact for plain rolling. Now that the rolling phase of the circle has been converted into flat rolling, the draft issue needs to be addressed [10]. The draft is defined as the reduction of rolling height. The initial and final heights were independent of the rolling system itself for flat rolling. In-ring rolling, though, the heights of entry and exit are combined as the height of exit in one rotation becomes the height of entry for the next rotation. This coupling effect can be given in terms of the advance mandrel speed and the system’s rotational speed [11]. When we consider the advanced instantaneous velocity $v_a = d_d/d_m$, and if the velocity of the mandrel is unchanged, the height shift in a single revolution can be interpreted as a finite difference $v_a = (h_1 - h_2)/t, n$, the period for a single rotation is determined from the ring and roll size and rotational velocity: $t = 60/n_{r1} = 60πd_d/v_d = 60πd_d/(d_d n_r)$.

Therefore, the height change can be described as

$$h_1 - h_2 = 60d_d/v_d(d_d n_r).$$

The $h_1$ and $h_2$ reflect the heights inside and outside the rolling area and the stress applied to the ring segment is proportional to the original sectional size, as there is no rotational annealing process. When we consider the peak draft state as the point of equilibrium of frictional and natural forces in the direction of rolling, a total approval angle for flat rolling can be given [12]. This condition is shown in Figure 4, where $F_n$ describes the normal force against the piece of work and $F_f$ the tangential frictional force to the move.

$$F_f \cos \alpha > F_n \sin \alpha;$$
$$F_f = \mu F_n > F_n \tan \alpha;$$
$$\mu > \tan \alpha.$$  

If we assume that the rolling radius exceeds the change in height (large rolling assumption),

$$\tan \alpha \approx \sin \alpha \approx (\Delta h/R)^{1/2}; \Delta h_{max} = \mu^2 R.$$  

This analysis is summarized in [4]. Setting these maximum drafts in the following relationship

$$\Delta h_{max} = \mu^2 d_d/2 = 60d_d v_{a max}/(d_d n_r);$$
$$v_{a max} = \mu^2 d_d n_r/(120d_d).$$

Therefore, to maintain the rotation of the ring during rolling, an upper limit is established on the prescribed mandrel advance velocity [2].

### 3.4 Numerical simulation

Figure 6 describes the modeling of ring rolling that includes workpiece, mandrel, and roller guide. Modeling is done using solids works and the mesh analysis is done in ANSYS. Wire modeling and meshing of ring rolling are given in Figure 5. The mesh domain is created for quality analysis.

Finite Element Analysis (FEA) is performed using the FEA software package called ANSYS. The computer package introduces and solves the formulas that control the action of elements. The material used for the process is Aluminium 6061 alloy [12]. The material properties are provided in Table 1.

The pressure of the roller against the workpiece could be either increased or decreased, based on the desired shape of the workpiece [8].

The roller moves around the workpiece that is fixed at the stationary state. The rotational speed of the roller is 40 rad/sec to get the desired dimensional output of the workpiece.

Figure 4 – Force balancing at critical rolling height

Figure 5 – The meshing of the ring rolling process
Table 1 – Properties of Aluminum 6061 alloy

<table>
<thead>
<tr>
<th>Property</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Density, kg/m³</td>
<td>2700</td>
</tr>
<tr>
<td>Brinell’s hardness</td>
<td>95</td>
</tr>
<tr>
<td>Rockwell hardness</td>
<td>40</td>
</tr>
<tr>
<td>Tangent modulus, MPa</td>
<td>1330</td>
</tr>
<tr>
<td>Ultimate tensile strength, MPa</td>
<td>310</td>
</tr>
<tr>
<td>Tensile yield strength, MPa</td>
<td>276</td>
</tr>
<tr>
<td>Young’s modulus, MPa</td>
<td>68.9</td>
</tr>
<tr>
<td>Ultimate bearing strength, MPa</td>
<td>607</td>
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<tr>
<td>Bearing yield strength, MPa</td>
<td>386</td>
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<tr>
<td>Poisson’s ratio</td>
<td>0.33</td>
</tr>
<tr>
<td>Fatigue strength, MPa</td>
<td>96.5</td>
</tr>
<tr>
<td>Shear modulus, MPa</td>
<td>26</td>
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<tr>
<td>Shear strength, MPa</td>
<td>207</td>
</tr>
<tr>
<td>Specific heat capacity, J/(kg·K)</td>
<td>896</td>
</tr>
<tr>
<td>Thermal conductivity, W/(m·K)</td>
<td>167</td>
</tr>
<tr>
<td>Elongation at break, %</td>
<td>12</td>
</tr>
</tbody>
</table>

4 Results

Rotational velocity is directly proportional to the time, which means that when the velocity is increasing the time taken for the process increases linearly. In this case, the rotational acceleration is equal to 50 rad/s².

From Figure 6a, it can be observed that the maximum and minimum static structural deformations for the process are 5.96 nm. Figure 6b describes the maximum static structural equivalent stresses equal to 0.027 MPa.

Figure 7a, b describes that deformations and stress increase with an increase in time. It can be observed from Figure 8 that the maximum static structural normal stress for the process is equal to 19.8 kPa.

5 Conclusions

The finite element analysis is done by considering aluminum 6061 alloys as workpiece material and structural steel as roller material in the ring rolling process. The values of normal stress, von Misses stresses, and strain energy are obtained. Fatigue power, mandrel life is accomplished by considering the number of cycles it could withstand. From the analysis, it is observed that von Misses stress of the work material is below the actual values. Workpiece strength is determined by changing the velocity of idle role and workpiece.
Моделювання та оптимізаційний розрахунок процесу прокатування кілець з конструкційної сталі сталі та алюмінієвого сплаву

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Анотація. Поточні дослідження проводилися за допомогою програмного комплексу ANSYS для оптимізації параметрів процесу прокатування кілець. Для оптимізації процесу прокатування були оцінені такі параметри, як швидкість, осьова подача ролика і параметри приводних роликів. Як підхід до оптимізації процесу було оцінено оптимальні значення параметрів та взаємозв’язки між ними. Параметри напружено-деформованого стану оцінювалися для різних швидкостей і сил, а також визначалися відповідні значення критичних параметрів. Для проведення дослідження було обрано конструкційні сталі та алюмінієві сплави, оскільки саме вони використовуються для виготовлення функціональних елементів і деталей коліс.

Ключові слова: валик, ANSYS, заготовка, пластичні деформації, еквівалентне напруження за Губером-Мізесом-Генкі, металевий сплав.